



# Modeling of Boundary-Value Problems of Heat Conduction for Multilayered Hollow Cylinder

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#### **Abstract**

The paper proposes the solution of the boundary value problem for the distribution of a non-stationary temperature field along the thickness of a multilayer hollow cylinder. The basis of the solution is the reduction method, the quasi-derivative concept, the modified Fourier method, and the problem of eigenvalues. The obtained analytical solution is modeled as a pseudocode and implemented on a specific numerical example.

#### **REQUEST OF THE PROBLEM AND ITS MATHEMATICAL MODEL**

The problem of the distribution of a non-stationary temperature field in the thickness of a hollow cylinder is reduced to a solution on the interval  $[r_0 \neq 0,$  $r_n$  of the differential equation:

 $2 \left( \begin{array}{c} 1 \\ 2 \end{array} \right)$ 

build 4) function u(r,τ): the  $u := (r, \tau) \rightarrow (1, 0) \cdot B \cdot P_0:$ 

5) find the characteristic equation the of problem for eigenvalues

 $\Phi := (\omega) \rightarrow (\det (P + Q \cdot B\omega))$ :

6) find the eigenvalues  $\omega_k$ :  $\omega_0 := 0$ : for i from 1 to k do  $\omega_{i}$ :=NextZero( $\omega \rightarrow \Phi(\omega)$ ,  $\omega_{i-1}$ ) end do:

7) assign the values of corresponding matrices for i from 1 to k do  $B\omega_i$ :=Matrix([[ $\beta_{11}$ ,  $B\omega_i$ :  $\beta_{12}$ ], [ $\beta_{21}$ ,  $\beta_{22}$ ]) end do:

8) assign the values of corresponding eigenvalues: for i from to 1 do

 $\begin{pmatrix} 1 \end{pmatrix}$ 

$$c\rho \frac{\partial t(r,\tau)}{\partial \tau} = \frac{1}{r} \frac{\partial}{\partial r} \left( r\lambda \frac{\partial t(r,\tau)}{\partial r} \right),$$

with the Newton's boundary condition

$$\begin{cases} \alpha_{0}r_{0}t(r_{0},\tau) - t^{[1]}(r_{0},\tau) = \alpha_{0}r_{0}\psi_{0}(\tau), \\ \alpha_{n}r_{n}t(r_{n},\tau) + t^{[1]}(r_{n},\tau) = \alpha_{n}r_{n}\psi_{n}(\tau), \end{cases}$$
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and the initial condition

$$t(r,0) = \varphi(r) \tag{3}$$

where  $t(r,\tau)$  – is a temperature, <sup>0</sup>C;  $t^{(1)}(r,\tau)=r\lambda t'_r$  – denotes a quasiderivative;  $\alpha_0$  and  $\alpha_n$  – coefficients of heat exchange between the fire medium and the cylinder surfaces, W/m<sup>2.0</sup>C;  $\psi_0(\tau)=345lg((8\tau/60)+1)+20$  – the law of changing the temperature of the fire medium (standard temperature mode);  $\psi_0(\tau)=20$  – temperature of the medium from the inside of the cylinder;  $\tau$  – time, s.

### **COMPUTER SIMULATION**

To solve the problem, we will conduct a computer simulation of a pseudocode solution for the computer algebra system

1) assign the values of the corresponding coefficients and notations of the problem:  $r_0:=0.2:$  $r_n := 0.5:$  c:=840: p:=2400:  $\lambda := 2.5:$   $\alpha_0 := 20:$   $\alpha_n := 4:$  $\psi_0 := \tau \rightarrow (345 \cdot \log 10 (8 \cdot \tau + 1) + 20) : \qquad \psi_n := \tau \rightarrow 20 : \qquad \phi := x \rightarrow 20 :$  $\alpha(\omega) := \sqrt{\frac{c \cdot \rho \cdot \omega}{\lambda}} : \alpha(\omega_{\underline{j}}) := \sqrt{\frac{c \cdot \rho \cdot \omega_{\underline{j}}}{\lambda}} :$ 

2) assign the values of the corresponding matrices and vector: Br:=Matrix([[1, (lnr-lnr)/ $\lambda$ ],[0,1]]): B:=Matrix([[1, ( $\ln r_n - \ln r_0$ )/ $\lambda$ ], [0,1]]): P:=Matrix ([ $[\alpha_0 r_0, -1], [0, 0]$ ]): Q:=Matrix([[0, 0],  $[\alpha_n r_n, 1]]$ ): 

$$R_{i}:=c \cdot \rho \int_{r_{0}}^{r_{n}} (1 \ 0) \cdot B\omega_{i} \cdot \left(\frac{1}{\alpha_{0}r_{0}}\right) dr \text{ end do:}$$
9) the norm square of the eigenvalues: for i
from 1 to k do
$$||R_{i}||:=c \cdot \rho \int_{r_{0}}^{r_{n}} (R_{i})^{2} r dr \text{ end do:}$$
10) the Fourier coefficients for development of the initial condition: for i from 1 to k do

$$f_{i} := \frac{c \cdot \rho}{||R_{i}||} \int_{r_{0}}^{r_{n}} (\varphi(r) - u(r, 0)) \cdot R_{i} r dr \quad \text{end do:}$$

11) the Fourier coefficients for expansion of the function  $u(r,\tau)$ : for i from 1 to k do  $u_{\underline{i}}:=\tau \rightarrow \frac{c \cdot \rho}{||R_{\underline{i}}||} \int_{m}^{r_{\underline{n}}} \left( \frac{\partial}{\partial \tau} \left( u \left( r, \tau \right) \right) \right) \cdot R_{\underline{i}} r dr \quad \text{end do:}$ 12) construction of the function  $v(r,\tau)$ : for i (\_

from 1 to k do 
$$v_i := \left( f_i \cdot e^{-\omega_i \tau} - \left( \int_0^t e^{-\omega_i (\tau - s)} u_i(s) ds \right) R_i \right)$$
 end do:  
13) construction of the solution  $t(r, \tau) :$   
 $t := (r, \tau) \rightarrow u(r, \tau) + \sum_{i=1}^k v_i$ :

14) result output in the form of the 3D-graph: plot3d(t(r,  $\tau$ ), r=r\_0...r\_n,  $\tau$ =0...3600)





The results of calculations can also be displayed in the form of explicit formulas, graphs, animations, tables, etc.

## **Conclusion**

The use of information technologies and methods of mathematical and computer modeling enables to quickly and clearly model certain phenomena or processes that arise during certain scientific researches. Using this approach significantly accelerates the time of research (compared with experimentally) and reduces the cost of scientific research.

The proposed direct method for calculating the temperature field in the plane structure can be used to study non-stationary thermal processes without the use of approximate and operational methods. Implementation of the results of such research has significantly reduced the time and amount of computing in determining the temperature distribution in plane structures, as well as improve the accuracy of the calculation in comparison with the approximate methods.